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TRANSLATOR'S AFFIDAVIT

I, Andrew Wilford, a citizen of the United States of America,  
residing in Dobbs Ferry, New York, depose and state that:

I am familiar with the English and German languages;

I have read a copy of the German-language document PCT appli-  
cation PCT/DE2005/000049 published 28 July 2005 as WO 2005/068100;  
and

The hereto-attached English-language text is an accurate  
translation of this German-language document.



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11 July 2006



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Method for Controlling the Position of a Mandrel of an Extrusion  
Press for Producing Tubular Workpieces

The invention relates to a method for controlling the  
5 position of a mandrel that is mounted in a hydraulic apparatus  
comprising a cylinder and a piston that form a piercing cylinder,  
of an extrusion press for producing pipes that are extruded from  
billets that are loaded into a holder mounted upstream from the  
extrusion die and pierced by means of the mandrel.

10 A metal extrusion press for the production of tubular  
workpieces and/or pipes has been disclosed in the German patent DE  
1,227,858. There, a mandrel is mounted with the piercing cylinder  
on the main extrusion piston. The piston of the piercing cylinder  
is connected to a piercing cross-bar that is guided in the platen  
15 of the press in a sliding manner. Alternatively to such inside  
punching devices, it is known to provide the piercing cylinder  
outside the main extrusion piston or cylinder of the press.

Since the mandrel in general can have varying lengths,  
the wear has to be taken into consideration, exact adjustment of  
20 the mandrel tip in the die opening is carried out by means of  
threaded spindles and spindle nuts with associated drives when  
extruding a pipe with a fixed mandrel. These are typically mounted  
in conjunction with mandrel stroke-limiting rods in the cylinder  
cross-piece or in the piercing cross-bar. Such an arrangement of a  
25 threaded spindle and spindle nut in the piercing cross-bar for  
limiting the stroke of the mandrel is disclosed for example in the  
patent mentioned above. The mandrel stroke-limiting rods here are  
fixed with one end in the cylinder cross-piece and carry stops on

the other end facing the pressure plate. The piercing cross-bar is supported against these stops with a nut that limits the stroke and consequently the mandrel and that can be adjusted with the threaded spindle.

5           In order to be able to position the mandrel guided through the ram in the tool or in the die when extruding tubular workpieces and/or pipes, and in order to maintain this position throughout the extrusion process with high precision, during the practical operation of the extrusion press the mandrel is held in  
10 position during operation by means of a piercing cylinder. To allow this position of the mandrel to be maintained in the die, the cylinder has to move the mandrel back at exactly the same speed at which the ram performs the forward movement. Here however disturbances due to forming forces, friction and hydraulic  
15 compressibility come into play, which the control system has to compensate for dynamically.

          Additionally, it is necessary to cover high speed ranges of 1:120 and greater. Finally, it is important to note that due to the application method considerable, variable tensile forces are  
20 applied to the mandrel, which can also reverse at the end of the extruding operation. In order to guarantee positioning, servo valves are used, via which the entire volume for the piercing cylinder is controlled. Since these servo valves can operate only a limited volume range, it is unavoidable that several servo valves  
25 in different nominal variables have to be provided parallel to each other for the speed ratio of 1:120.

          It is therefore the object of the invention to create a method of the type mentioned above that enables mandrel control

that meets all necessary requirements in a simple manner and particularly without requiring servo valves.

This object is achieved according to the invention in that the piercing cylinder is directly driven by pumps that are  
5 adjusted to a defined pumping volume as a function of the extrusion speed and that a further pumping volume is added to the previously computed pump conveying volume, wherein for the purpose of controlling the position of the mandrel a control valve acting upon the front ring compartment of the piercing cylinder is connected to  
10 a sump. As a result of the direct drive, i.e. without an interposed control valve, so that no servo valve is mounted between the pump and piercing cylinder, but rather only conventional, cost-efficient and inexpensive-to-operate cartridge valves are used for the direction of motion, no pressure loss is produced for the pump  
15 volume flow. This way it is possible for the pump pressure to correspond to the operating pressure at the piercing cylinder. In addition, almost no energy losses occur, and the operating pressure at the pump is lower. The direct drive necessitates only a single, small control valve for the entire speed range, which valve  
20 additionally operates with very high precision and very quickly. This is associated with very large cost savings.

With the control principle according to the invention, the pumps are adjusted to a pumping volume as a function of the extrusion speed, which volume produces a substantially equal  
25 retraction speed of the piercing cylinder. This way, the mandrel assumes a substantially constant position in the die throughout the entire extrusion process. In order to enable the positioning and the correction of disturbances at the same time, according to the

invention an additional pumping quantity is added to the pump conveying volume, over and above the computed pumping volume of the piercing cylinder. This additional pumping volume prevents the piercing cylinder from moving rearward out of the die position against the forward extrusion direction. The small control valve provided for control connected the front ring compartment of the piercing cylinder establishes the connection between the ring surface of the cylinder to the sump and controls the oil quantity to the sump required for positioning. If the oil volume to the sump is less than the value of the additional pumping volume, the mandrel is moved rearward out of the die; if the oil volume to the sump is higher, the mandrel is moved forward into the die. The control valve that maintains the position by means of a controller thus balances disturbances.

According to a preferred embodiment of the invention, it is proposed that the outlet pressure of the piercing cylinder is adjusted to a defined pressure. This way, control of the mandrel position can be achieved also with decreasing tensile forces in the mandrel or in the event of a reversal of the forces. To this end, advantageously a proportional pressure control valve is connected to the rear compartment or the piston side of the piercing cylinder. This way, stable control can be achieved even with reversing or decreasing forces on the mandrel.

If it is provided in an advantageous embodiment that the pressure levels in both compartments of the piercing cylinder are monitored, for example by means of pressure load cells connected to both the front ring compartment and the rear compartment of the piercing cylinder, in the event the value drops below a defined

value the integration of a second controller and the monitoring of the pressure levels in both compartments allow the outlet pressure to be increased enough so that the defined pressure is present in the retraction side of the piercing cylinder. As a result, the hydraulic system is always in the tensioned state and allows a control regardless of the direction of the force.

Further characteristics and details of the invention are disclosed in the claims and the description provided hereinafter with reference to the schematic illustration of a control concept shown in the sole figure.

The drawings shows in a diagrammatic illustration of a standard extrusion press 1 that is used to produce tubular workpieces and/or pipes 2, only the tool 3 comprising a die, the holder 5 that is mounted upstream and receives a billet 4 to be extruded, a ram 6 with a cross-piece 7 and a piercing cylinder 8 with a hydraulic system. The piercing cylinder 8 has a piston 9 that can be displaced forward and rearward, with a mandrel 10 that is carried by the piston, guided through the ram 6 and positioned with its tip or front end in the die 3.

The piercing cylinder 8 is driven directly by pumps 11. For this purpose, the front ring compartment 12 is connected via a hydraulic line 13 to the illustrated pump 11 that is associated with an intake valve 14 in the pumping direction. A sump line 15 branching off the hydraulic line 13 comprises a small, integrated control valve (NG10) 16 that is connected to an unillustrated controller and empties into a sump 17.

For controlling the position of the mandrel 10 with exact positioning of the front end in the tool 3, the pumping volume of

the pump 11 that has been previously computed as a function of the extrusion speed is supplemented by an additional quantity of hydraulic fluid fed to the front ring compartment 12 of the piercing cylinder 8 in order to correct disturbances. Operation of the small control valve 16 effected by the controller at the same time to move the mandrel 10 rearward out of the die 3 when the oil volume to the sump 17 is less than the value of the additional pumping volume. On the other hand, the mandrel 10 is moved further into the tool 3 when the oil volume to the sump 17 is greater than the value of the additional pumping volume. As a result, the mandrel 10 always assumes a substantially constant position in the tool 3 throughout the entire extrusion process.

The two compartments 12 and 18 of the piercing cylinder 8 are monitored in terms of pressure. For this purpose, a pressure load cell 19a or 19b is associated both with the front ring compartment 12 and the rear compartment 18. In an outlet line 20 leading from the rear compartment 18 of the piercing cylinder 8 to the sump line 15 and connected thereto via the small control valve 16, a proportional pressure control valve or pressure control valve 21 is connected to another controller that is not shown.

By monitoring the pressure levels in both compartments and switching the pressure control valve 21 by means of the second controller, in the event that the defined pressure level is not reached the pressure in the rear compartment 18 can be increased enough so that the defined pressure is present in the front ring compartment 12 or the rear compartment of the piercing cylinder 8. The hydraulic system is therefore in a constant tensioned state and allows a control regardless of the direction of the forces, so that

a response to decreasing tensile forces on the mandrel 10 or a reversal of the forces is possible.